



PENN  
**HYPERLOOP**

# NoDiggity V2

Overall goal was to dig a 30m long horizontal tunnel of 0.5m diameter. I was the Cutterhead and Main Drive systems RE (responsible engineer) for a system that generated 2.2 kNm of torque. Was the de facto mechanical-side team lead and coordinated between the propulsion system, soil removal and ground conditioning systems, to ensure delivery on the machine.

Not mentioned in this portfolio are the numerous hours (~50% of my time) spent planning the logistics of moving and testing ops, organizing a temporary build space, and fighting with school management for recognition, and fundraising talks.

**Official dig length of 1.5m to win Rookie Award with smallest team and simplest TBM design (as per judges).**



# Excavation Systems - NoDiggity V2 - Penn Hyperloop

Meain Drive Torque Calculation  
Utilizing the findings of Hu's 2011 paper\* we can split up our torque contributions as seen below.

| Torque Contribution   | Torque (kNm) |
|-----------------------|--------------|
| T1 (Front)            | 1.169910308  |
| T2 (Lateral)          | 0.5114362    |
| T3 (Back)             | 0.389970103  |
| T5 (Opening Shear)    | 0.13847143   |
| T6 (Agitating torque) | 0.2662796    |
| Total Torque          | 2.476067639  |

Using a FOS of 2 we get a recommended Torque of ~5 kNm

Key System Metrics: (Rossi Group motor: HB3 132 S 4, Rossi Group planetary gear reducer: EP R 3EL)

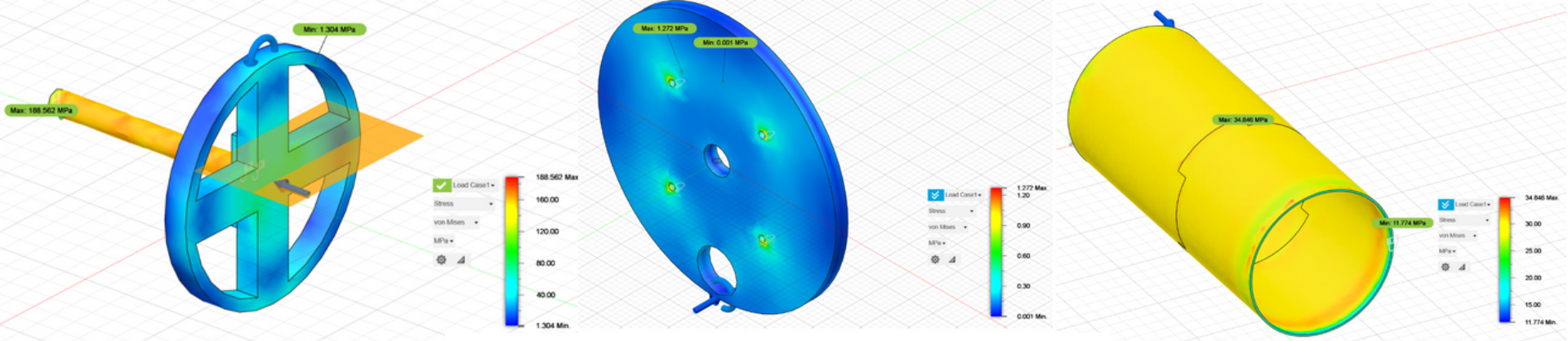
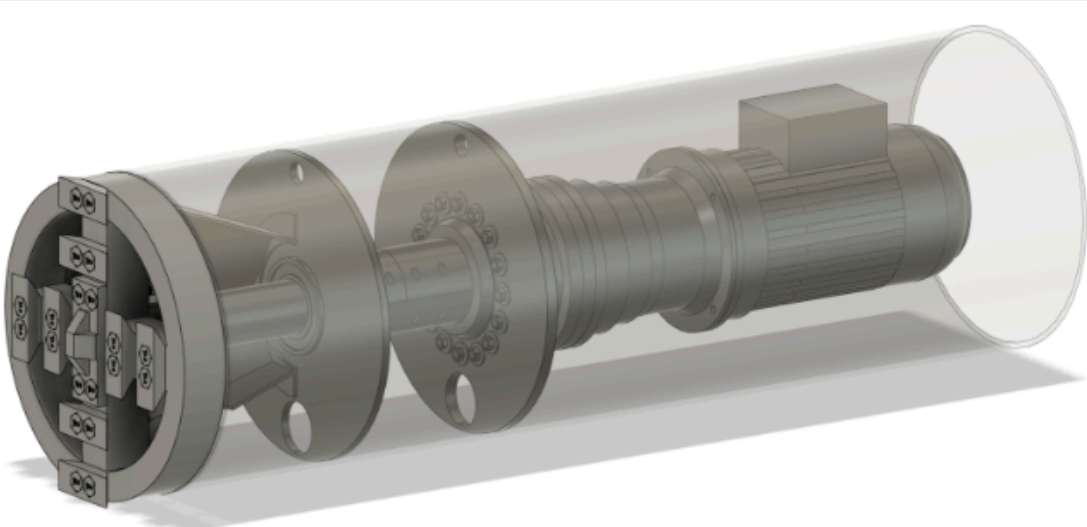
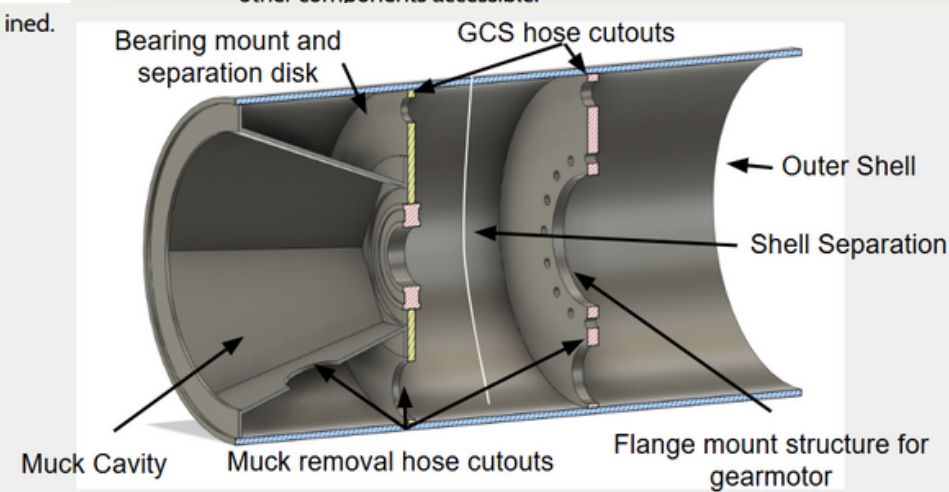
- Estimated Output Torque: 4780 Nm
- Nominal Output Speed: 8.1-11.9 rpm
- Mass: 221 kg
- Effective reduction ratio: 142.56-209.44
- Gearmotor configured for expected axial/thrust loads from manufacturer.

Coupling:

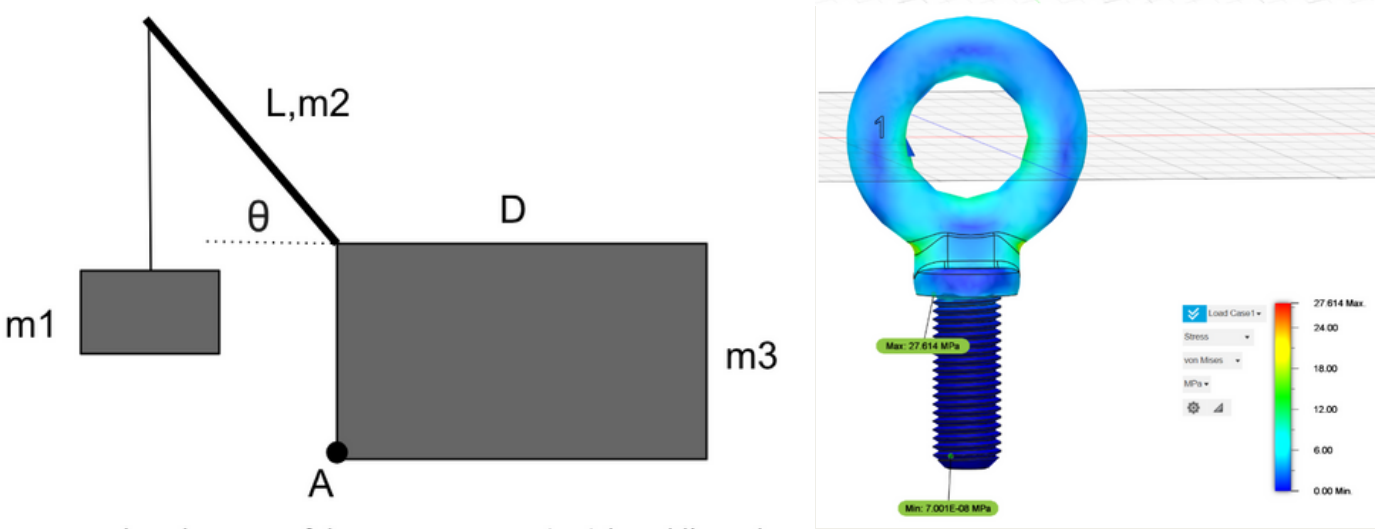
- Custom machined keyway two-piece coupling (Stafford)
- 316 Stainless Steel (for corrosion protection)
- 6 screws on each side
- Holding Torque: 3723 Nm

Mounting Plate:

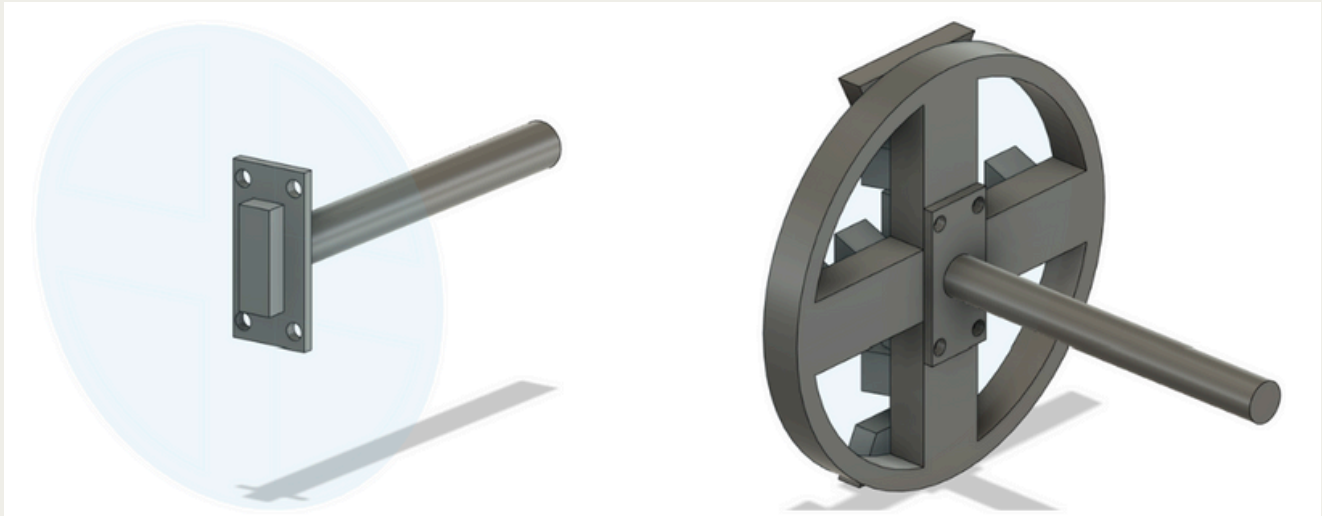
- Assuming AISI 4340 242 HR steel with a Young's modulus a plate thickness of of 33mm would be needed to support the system. Considering a safety factor of 1.5, a plate thickness of 49.5mm (50mm) would be desirable.
- Mounting Strategy: Weld ring plate to outer body casing to create a flange mounting system for plate to be bolted to. This allows for removal of plate to make other components accessible.



**Improved design:**  
Overall structural analysis shown for various failure modes.



Added: lifting points, alignment peice to achieve concentricity and better torque transfer, switched out gearmotor based on updated assumptions.



|    | A                                          | B           | C | D | E | F | G | H | I | J | K | L |
|----|--------------------------------------------|-------------|---|---|---|---|---|---|---|---|---|---|
| 1  | <b>Torsional Shear</b>                     |             |   |   |   |   |   |   |   |   |   |   |
| 2  | Applied Torque (Nm)                        | 2700        |   |   |   |   |   |   |   |   |   |   |
| 3  | Shaft Radius (m)                           | 0.0508      |   |   |   |   |   |   |   |   |   |   |
| 4  | Weld thickness (m)                         | 0.00635     |   |   |   |   |   |   |   |   |   |   |
| 5  | Polar moment of inertia (m <sup>4</sup> )  | 0.000006274 |   |   |   |   |   |   |   |   |   |   |
| 6  | FOS                                        | 5           |   |   |   |   |   |   |   |   |   |   |
| 7  | Torsional shear stress (N/m <sup>2</sup> ) | 109311612.5 |   |   |   |   |   |   |   |   |   |   |
| 8  |                                            |             |   |   |   |   |   |   |   |   |   |   |
| 9  | <b>Normal Shear</b>                        |             |   |   |   |   |   |   |   |   |   |   |
| 10 | Applied Force (N)                          | 265748.0    |   |   |   |   |   |   |   |   |   |   |
| 11 | Weld area (m <sup>2</sup> )                | 0.001170    |   |   |   |   |   |   |   |   |   |   |
| 12 | Normal shear stress (N/m <sup>2</sup> )    | 227098227.4 |   |   |   |   |   |   |   |   |   |   |
| 13 |                                            |             |   |   |   |   |   |   |   |   |   |   |
| 14 | Filler material required max stress (psi)  | 36554.6574  |   |   |   |   |   |   |   |   |   |   |
| 15 |                                            |             |   |   |   |   |   |   |   |   |   |   |
| 16 | <b>Online model (w/ safety factor)</b>     |             |   |   |   |   |   |   |   |   |   |   |
| 17 | Weld fillet radius (m)                     | 0.003175    |   |   |   |   |   |   |   |   |   |   |
| 18 | Filler material required max stress (psi)  | 43053.69067 |   |   |   |   |   |   |   |   |   |   |
| 19 |                                            |             |   |   |   |   |   |   |   |   |   |   |
| 20 | ER 4340 max stress (psi)                   | 168,000     |   |   |   |   |   |   |   |   |   |   |
| 21 | FOS                                        | 3.902104497 |   |   |   |   |   |   |   |   |   |   |
| 22 |                                            |             |   |   |   |   |   |   |   |   |   |   |
| 23 |                                            |             |   |   |   |   |   |   |   |   |   |   |
| 24 |                                            |             |   |   |   |   |   |   |   |   |   |   |
| 25 |                                            |             |   |   |   |   |   |   |   |   |   |   |
| 26 |                                            |             |   |   |   |   |   |   |   |   |   |   |

Effective weld Area (= Throat x Length)  
Effective Weld Length  
Effective weld Throat (= Leg x 0.707)

$J = 2 \pi a (r + a / 2)^3$

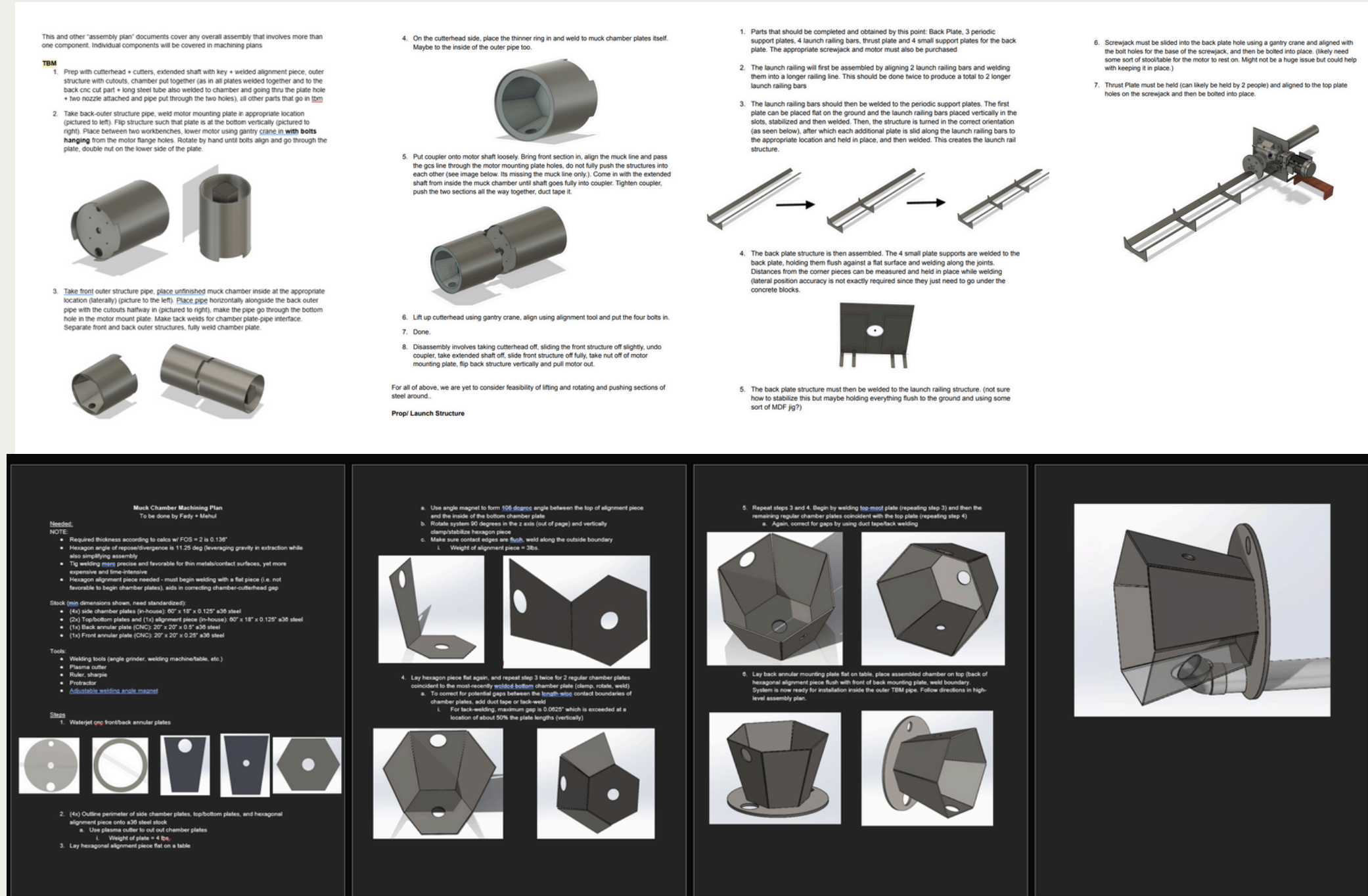
$\tau = \frac{2.83 M_t}{h D^2 \pi}$

$\tau$  = Shear Stress, MPa (psi)  
 $M_t$  = Torque N-m (lbf-in)  
 $D$  = Diameter, m (in)  
 $h$  = Size of Weld, m (in)  
 $\pi$  = 3.14157 (PI)

**Initial Design** – Started off with main physics requirement of torque needed. Accordingly, sourced sized a gearmotor + custom coupling system that fits into CAD modeled outer structure.



**Assembly and Machining Plans generated:**  
(Test Plans were skipped due to lack of funding  
early enough into our process.)



**Drawings** – Large plates plates were sent out for waterjet/ laser CNC cutting. Rest were machined in-house.



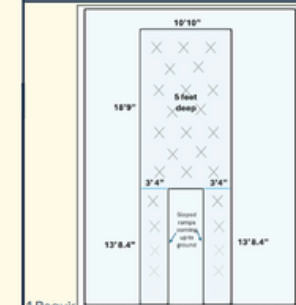
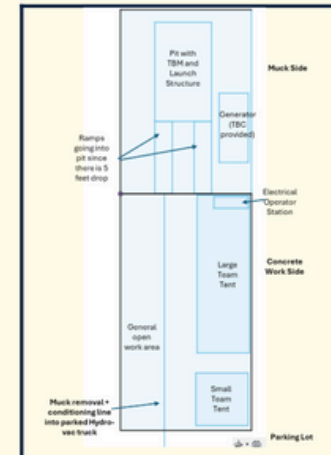
# Excavation Systems - NoDiggity V2 - Penn Hyperloop



Section 3

## Launch Setup

### Site Layout



1. Required from TBC on Wednesday/Thursday  
a. Excavator + operator on setup day to dig pit (10'10" x 10'10" x 5') and ramps. **Needed at earliest possible time** as soon as site opens.  
b. Two pallet jacks for team to move pallets.  
c. Generator placed next to pit
2. Required from TBC on Wednesday/Thursday  
a. Telehandler or excavator + operator to lift propulsion structure and TBM separately.  
b. Retrieval pit to be dug 10 meters from start point. Same dimensions as start pit.

### Retrieval Plans

- Retrieval pit is dug for our max possible dig length of 11 meters. Once TBM surfaces, all pipes/wires will be disconnected, removable eye bolts will be screwed into the top and lifted out.
- In the event of machine failure before finishing dig, excavator will be required to dig to the depth of tunnel crown, team will dig around the pit to reveal full TBM for lifting.

Section 2

## Cutterhead & Main Drive

### Biggest Risks

1. Reaction torque causes TBM to spin  
a. Mitigation: Gearmotor can spin in both directions. Will switch if IMU indicates excessive TBM rotation  
b. Unlikely to happen since TBM is heavy + friction between back of TBM and clay pipe resists rotation
2. Outer scrapers tool wear  
a. An overcut is present where the outer cutters furthest point leads to an effective 20.5" OD (above the 20" of TBM itself)
3. Leakage of oil from venting hole on gearmotor  
a. Mitigation: Oil leakage point is placed facing up, gravity will not act to cause a leakage.
4. Motor mounting plate-outer structure weld failing (since made by amateur student welders)  
a. Failure does not harm humans. It is safe.

### Subsystem Single Points of Failure

1. Weld at shaft-alignment plate interface fails  
Both parts are made of stronger 4340 steel + professionally TIG-welded with appropriate 4340 wire.
2. Coupler fails  
Unlikely since coupler is rated for max-torque expected (with FOS). This torque is likely not required at initial stages of dig either.

### Open Action Items

1. Fasten cutters on cutterhead  
Done at comp to minimize cutter damage.
2. Mount motor into TBM structure  
Will be done on Monday, Tuesday latest
3. Mount coupler, shaft, cutterhead onto TBM  
Will be done on Monday, Tuesday latest
4. Spin Test:  
Will be done once TBM is assembled and high power on is approved

Evaluated mining readiness across mechanical systems to present to overseeing organization (Boring Company)



Manufacturing + Assembly at Nextfab  
(makeshift workspace for the machine)

Learnt and applied flux-core MIG Welding up to 1/4" thickness on Mild Steel.

Coordinated site prep, lifting ops,  
troubleshooting dig/ demo day failures